

DAVID W. TAYLOR NAVAL SHIP RESEARCH AND DEVELOPMENT CENTER

Bethesda, Md. 20084

CAVITATION PERFORMANCE OF PROPELLERS WITH AND WITHOUT CUPPING

by

J. G. PECK and B. L. FISHER

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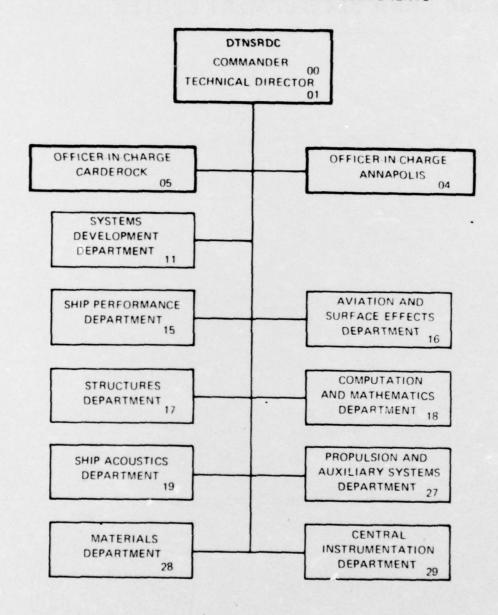
SHIP PERFORMANCE DEPARTMENT



SPD-725-01

September 1976

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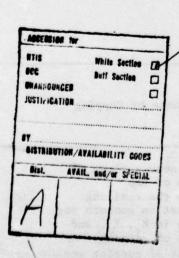
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20. ABSTRACT (Continued)

the resulting increase in effective pitch. It is concluded that cupping is an effective means of correcting an underpitched propeller, at the expense of efficiency and danger of increased cavitation.



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NOTATION

A _E	Expanded area of propeller blades (ft^2) , (m^2) $A_E = EAR (A_0)$
Ao	Disc area of propeller (ft ²), (m ²)
	$A_{O} = \frac{\pi D^{2}}{4}$
Ap	Projected area of propeller blades A _p = A _E (1.067-0.229 P/D)
C	Blade section length (ft), (m)
C _{0.7}	Blade section length at 0.7 radius (ft), (m)
D	Propeller diameter (ft), (m)
EAR	Expanded area ratio A _E /A _O
g	Acceleration due to gravity (ft/sec ²), (m/sec ²)
h	Propeller submergence (ft), (m)
J	Advance coefficient J=V/nD
K _T	Thrust coefficient $K_T = \frac{T}{\rho n D^4}$
K _T /J ²	Loading coefficient
K_{T} K_{T}/J^{2} K_{Q} K_{Q}/J^{3}	Torque coefficient $K_Q = \frac{Q}{\rho n D^5}$
K _Q /J ³	Powering coefficient
n	Propeller rotational speed (rev/sec), (r/s)
P	Propeller pitch (ft), (m)
P/D	Pitch-diameter ratio
PA	Atmospheric pressure (1b/ft ²), (N/m ²)
PH	Static water pressure, P = pgh, (1b/ft ²), (N/m ²)
P _V	Vapor pressure (lb/ft ²), (N/m ²)
T	Propeller thrust (1b), (N)

Q Propeller torque (1b/ft), (N·m)

Torque load coefficient, Qc

 $Q_c = \frac{2.55 \text{ K}_Q}{(\text{J}^2 + 4.84) (\text{EAR}) (1.067 - 0.229 \text{ P/D})}$

Velocity of boat (ft/sec), (m/sec)

Resultant velocity of water at 0.7 radius of vo.7 propeller (ft/sec), (m/sec)

 $v_{0.7}^2 = \frac{J^2 + 4.83}{J^2} v^2$

Amount of cupping at 0.7 R, (in), (m) x

Propeller open water efficiency, n

 $\eta = \frac{K_T}{K_Q} \frac{J}{2\pi}$

Additional pitch angle required for new pitch, degrees 0 A

New required pitch angle, degrees

Original pitch angle, degrees

Mass Density of water (1b-sec²/ft⁴), (K_g/m^3) Cavitation number, $\sigma = \frac{P_A + P_B - P_V}{(1/2)\rho v^2}$

Local cavitation number, $\sigma_{0.7} = \frac{P_A + P_H - P_V}{(1/2)\rho v_{0.7}^2}$ ⁰0.7

Thrust load coefficient, $\tau = \frac{T}{(1/2)\rho A_p v_0}$

ABSTRACT

Four commercial propellers were characterized over a range of cavitation numbers and advance coefficients. Three of the propellers were then cupped to different degrees on the trailing edge and characterized over the same range of cavitation numbers and advance coefficients. The results show an increase in K_T , K_Q , and effective pitch corresponding to increasing degrees of cupping. An empirical relationship is derived between the amount of cupping and the resulting increase in effective pitch. It is concluded that cupping is an effective means of correcting an underpitched propeller, at the expense of efficiency and danger of increased cavitation.

ADMINISTRATIVE INFORMATION

This work was performed for Naval Ship Engineering Center, Norfolk Division under Project Order No. N64281-76-PO-6-0005.

INTRODUCTION

Cupped propellers have been manufactured for a number of years and they have become popular for small and medium size pleasure boats. Most manufacturers of small propellers offer one or more of their propeller styles with varying degrees of cupping. Improved performance is claimed for these cupped propellers. Recent full scale trials of the 65' MK III patrol boat appear to support this claim. A significant increase in speed was realized for this craft when a small amount of cupping was applied to her existing propellers. Power measurements prior to cupping, however, indicated that the propellers were not absorbing the full engine power at maximum engine RPM. After cupping these propellers, the maximum available power of the engines was absorbed at the maximum engine RPM, and craft speed increased. It is likely that the increase in power absorbed by the propellers in this case resulted from an effective increase in pitch caused by the cupping.

The cupping of propellers is considered to be an art in the propeller industry. The process (i.e, cupping) increases the effective pitch through deflecting the trailing edge of the propeller-blades (which also increases the trailing edge blade camber). In order to determine the effect of various amounts of cupping on the performance of commercial propellers a limited experimental program was under taken at DTNSRDC. The characteristics of four commercial propellers were determined with and without cupping. The results are reported herein.

The geometry of each propeller along with the range of experimental parameters are shown in Table 1. A photograph of one of these propellers is shown in Figure 1.

DTNSRDC	Nominal	Mode1	Expanded	Number
Propeller	Pitch-Diameter	Propeller	Area	of
Number	Ratio	Diameter/ins (m)	Ratio	Blades
4685	1.0	12(0.3048)	0.694	3
4686	1.0	12(0.3048)	0.582	3
4687	1.1	12(0.3048)	0.593	3
4688	1.3	12(0.3048)	0.723	3
DTNSRDC	Geometrical	Cavitation	Advance	
Propeller	Changes	Number	Coefficient	
Number		Range		
4685	None	5.8-0.5	0.55-1.05	
4685	Cupped	5.8-0.5	0.60-1.05	
4686	None	5.8-0.5	0.55-1.05	
4686	Cupped	5.8-0.5	0.55-1.05	
4687	None	5.8-0.5	0.60-1.15	
4687	Cupped	5.8-0.5	0.60-1.15	
4688	None	5.8-0.5	0.65-1.40	

EXPERIMENTAL PROCEDURE AND RESULTS

Propeller open-water characteristics of the four propellers were obtained in the Center's deep water towing basin. The data were reduced to the usual non-dimensional coefficients of thrust and torque and are presented in Figures 2 and 3. Reynolds number during the open-water characterization varied from 9.0×10^6 to 11.0×10^6 .

Cavitation characteristics of the propellers were obtained in the 24-inch variable pressure water tunnel. Tunnel water velocities for each uncupped propeller were established by setting thrust values in the water tunnel equal to the thrust values obtained in open water at the same advance coefficient. Tunnel pressures were adjusted to cover a range of cavitation numbers from 5.8 to 0.5. These cavitation numbers represent a range of ship speed from 12 to 40 knots.

After the uncupped propeller experiments were completed three of the propellers were cupped in the DTNSRDC propeller shop. The blade outline and thickness of these propellers made it difficult to cup them at radii less than 0.5 or greater that 0.9. As a result the cupping is maximum at 0.7 radius and fairs to zero near the hub and at the blade tip. Propellers 4685, 4686 and 4687 were cupped in different amounts designated respectively as heavy, light and medium cupping. Cavitation characteristics of the cupped propellers were obtained over the same range of cavitation numbers and advance coefficients as the uncupped propellers. The thrust and torque data obtained from the cavitation experiments were reduced to the usual nondimensional coefficients, K_T and K_Q . In addition, efficiencies

(n), thrust loading coefficients (K_T/J^2) and τ_c and torque loading coefficients (K_Q/J^3) and τ_c were calculated from faired values of thrust and torque coefficients. All the force coefficients are given in Tables 3 through 8.

Curves of the cavitation performance of the propellers are presented in Figures 4 through 14. Sketches of the extent of cavitation on the propeller blades are shown in Figure 15 for the heavily loaded conditions and in Figure 16 for the lightly loaded conditions.

DISCUSSION

When the curves of cavitation performance are treated as typical propeller series data, values of effective pitch may be assigned to the cupped propellers based upon their thrust producing capability. These results are given in Table 2 below:

TABLE 2

Effective Pitch Ratio of Propellers Based on Thrust Performance

DTNSRDC	Geometrical	Effective
Propeller	Changes	Pitch Ratio
Number		
4685	NONE	1.00 .
4685	Cupped	1.15
4686	NONE	1.00
4686	Cupped	1.05
4687	NONE	1.10
4687	Cupped	1.18
4688	NONE	1.30

By correlating the change in effective pitch of these propellers and the amount of cupping done to them, an empirical relationship has been established to calculate the degree of cupping needed to accomplish a predetermined change in effective pitch.

If one associates cupping with the added deflection of the blade trailing edge after cupping (the deflection being measured perpendicular to the nose-tail-line of the blade, at the 0.7 radius), cupping may be defined as:

(See Figure 17)

$$x = (1/2)C_{0.7} \tan \theta_A$$

where

x = amount of cupping at 0.7 R

Co.7 = blade chord length at 0.7 R

θ = the approximate change in pitch angle between propeller pitch and desired propeller pitch.

If p_n and θ_n are the new pitch and pitch angle desired, where $\theta_n = \tan^{-1} \frac{p}{(2\pi r_{0.7})}$

$$\theta_{A} = \theta_{n} - \theta_{o} = \tan^{-1} \left(\frac{P_{n}}{2\pi r_{0.7}} \right) - \tan^{-1} \left(\frac{P_{o}}{2\pi r_{0.7}} \right)$$

where θ and P are the original pitch and pitch angle.

It was found that the proper cupping can be achieved by using a ball or curved anvil. The radius of the ball may be determined by using the following equation (See Figure 18 for definition):

$$r = \frac{x^2 + y^2}{2x}$$

where

r = radius of desired ball

x = amount of cupping at 0.7R

y = 10% of blade chord length at 0.7R

A numerical example for these computations is shown in Appendix

A. In cupping the propellers the center of the ball should be
positioned at 90% of the chord length from the leading edge of the
blade, starting at the hub and progressing toward the tip to 0.9R.

From 0.9R to 0.95R the center of the ball moves linearly from 90% of

the chord length to 100% of the chord (i.e., to the trailing edge).

Propeller cupping is a practical method for increasing the effective pitch of an existing propeller. However, there are two potential problem areas. One problem is that the cupping produces a blade trailing edge which is susceptible to trailing edge cavitation. This effect is illustrated by the sketches of cavitation present, under typical operating conditions, on these propellers (See Figures 15, 16, and 17). Also cupped propellers, in general, seem to have a somewhat lower efficiency than propellers designed specifically for the desired pitch. Thus cupping is recommended only when the existing propeller is underpitched and the required higher pitch propeller is not readily available.

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CONCLUSION AND RECOMMENDATIONS

An approximate definition of cupping was derived from data obtained using four commercial propellers. It appears that the cupping of an in-service propeller is an effective measure to correct the initial mis-match between the propeller and the powering system. A properly cupped propeller will absorb the available power at the expense of lower efficiency and with the possibility of an increased amount of cavitation.

Based on this limited sample (only four propellers were evaluated), it is recommended that cupping only be used as a corrective measure.

The empirical relationship between cupping and effective pitch, in a strict sense, is valid only for the type of cupping described in this report. If a different method is used to achieve cupping, the conclusions drawn from these experiments may not be valid. A much larger sample of commercially cupped propellers would be required if one desires to arrive at a "general rule of thumb".

APPENDIX A

The procedure to follow in order to determine the amount of cupping needed to raise the effective pitch of an existing propeller to some new value is as follows:

1. Estimate the desired pitch ratio and calculate the new pitch. For this pitch, the new pitch angle, θ is

$$\theta_{n} = \tan^{-1} \left(\frac{P_{n}}{2\pi r_{0.7}} \right)$$

- 2. Calculate the required additional pitch angle θ_A : $\theta_A = \theta_n \theta_o$, where θ_o is the original pitch angle.
- 3. The trailing edge of the propeller should be cupped, at the 0.7 radius, an amount

$$x = (\frac{C_{0.7}}{2}) \tan \theta_A$$
 [in]

For example consider a propeller of:

$$D = 2.5 \text{ ft } (.762 \text{ m})$$

P/D = 1.0

$$C_{0.7} = 1.458 \text{ ft } (.444m)$$

How much should this propeller be cupped to achieve a desired P/D = 1.1?

The original pitch is P = (D)(P/D) = (2.5 ft)(1.0) = 2.5 ft

The original pitch angle is $\theta_0 = \tan^{-1}(\frac{P}{2^{\pi}r_{0.7}}) = \tan^{-1}[\frac{2.5}{(2)(\pi)(.87)}] = 24.453^{\circ}$

The new pitch is P = (2.5)(1.1) = 2.75 ft

The new pitch angle is $\theta_n = \tan^{-1}(\frac{P}{2\pi r_{0.7}}) = \tan^{-1}[\frac{2.75}{(2)(\pi)(.87)}] = 26.706^{\circ}$

Therefore:

$$\theta_{A} = \theta_{n} - \theta_{o} = 26.706 - 24.453 = 2.253^{\circ}$$

Then the amount of cupping needed, x, is

$$x = (\frac{C_{0.7}}{2}) \tan \theta_A = (\frac{1.458}{2}) \tan 2.253^\circ = .029 \text{ ft} = .344 \text{ inches (.009 m)}$$

The ball radius needed to effect this amount of cupping is:

$$r = \frac{x^2 + y^2}{2x}$$

where x = .344 inches
y = 10% of blade chord at 0.7 R=1.749 in (.0444 m)
$$r = \frac{(.344)^2 + (1.749)^2}{2(.344)}$$
$$= 4.62 in (.1174 m)$$

Table 3

Cavitation Performance Characteristics of Propeller 4685 Cupped

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Cavitation Performance Characteristics of Propeller 4687

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	100001	. 5 211	. 706			1808.	.2673	.273					1000201	****		5382	*5.2.	.7879	.2420		.1765	, , ,											
1112000	***	.7699	. 2488	. 2197		1 1 2 2	.1167	0260				77 282	1001	. 05.07		. 101.	1811.	.1194	2001.		2.00												
	,	9009.	2007.				96 36.	1.00.1	1.1500	1.1488			,	****	1986		::	98 96	00.6.		1.1980	1.1.00											
	7.MUC	:141.	.151.	. 1 350	1000	. 0830	. 967	0250	. 9748	.010.			1100	.1985	9521.		.1734	9460	. 0776		2010.				TAUC	****	. 0781	. 0399		5440.	. 9382	. 416.	. 000.
;	\$16me7		. 5337	.609			. 9115	2566	1.1600	1.2447		. 1.5	STGPAT	.1 83 8	1204	2951.	28.	. 2150	. 235.7		. 3000	6141.		*	SIGHAT		040	. 9584	6999	. 8786	9580	1001	.1673
. 16.1	36	***	1910.	. 9351	9274	. 9241	. 9299		. 9116	1866.		21644	36	. 9284	0110	. 9110		. 3251	.1214		.010	. 100.		STGRA	36		. 010	.0125	.0160	5916.	. 9179		. 9930
1.100	KQ/33	1642.	.1345	9666			. 0310	. 0230		. 3073		1.100	KQ/J3	11513	.1402	. 9463	.0754	.0631	.0310	9530	6016.	. 1066		1.100	EC.13	****	. 0343	.020.	1920'	. 1215	.0167	8400	1200.
PITCH 047T0 .	25/13	5002	****	.3661	2367	.1636	.1207		. 6336	**16.		# 01164 PATE	25/13	.4305	*****	. 3575	****	.1984	5611.	1001	9246	26.20		PITCH 84710	51,12	****	.0862	.0926	1960.	66.0	.0630	6820.	1100.
	FFFTC	2864.	5474	.585.		1629.	.6295	2465	1681	.3137		1991	21443	6254.	5016	. 8 40.	2019.	. 8875	9669.	2000	. 6241	1866		14 1894:	21443	1636	1997	1864	. 5189	2265	6494		*000.
PROPELLER NUMBER	100030	5385	1194	0024			.2654	***		.1110			100001	.3266	. 1858	2984		.3146	.2724		-	.1010			100001	. 290		1513	5271.	1961	11.60	8068	21.0.
	1,001	2810	.2310	.2060		.1325	.1084		. 0489	.0190		11126	*1001	.1950	.1766	1162	.1910	.1911	.1259	. 0750	. 9516			7734084	TUC1#		2240.	1650.	.0687	.0721	. 06 30	.0253	4100.
	,	***				****				1.1500			,	.6880	. 65 88	. 7500		.4000	****						,	****			. 6500	94 86	1.9000		1.1.200

: IIIIIIIIIII

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Table 7

Cavitation Performance Characteristics of Propeller 4687 Cupped

	1 100		191	51.			. 077	290.						1800	. 051							?!													
:	SIGNAT	.545	.3124	186			. 5137	. 5966						SIGNAT			781	9200		.1179	1 361														
	20	****	21.70	.0395	. 0 3 5 6	1820.	. 3247	. 126.				******		20	. 916.	9116	. 9233	1920.	.3254	2920.	. 3186	. 9157													
1.100	86/5%	2705.	.1227	1868.	.0715	. 1417	. 6319	. 0243	2010.	•		PTC- 0110 - 1.00		£6/03		.0707	1998.		****		1120.	.0158													
- 1104 84710 -	KT/ 32	6459.	.4389	. 3514	9822	.1600	.1278	.0990	2007			PTTC# 8811		*17.32	*682*	2002		5602.	1111		. 9829	1090.													
	CFF 1C	. \$030	. 5696	. 5970		1000	. 6373	.6223	. 5428					21.43	. 3928		\$116	2665	1919.	. 6307	.6237	5409.													
PROPELLER NUMBER . 1687	100001		. \$175	.4796		.3578	.3193	2102.	9116			***** ********************************		100001	28.41.	.2078	.2787	***	.3234	.3081		7812.													
PROPELL	#1001	.2766	5462	6422.	. 2000	1516	.1270							*1004 M	.1754	. 0056	.120	1 341		.1285		.0729	. 1623												
	,		.7580				1.0000	1.050						,	****		. 7.00			. 9400			1.1500												
	auc.	.2143	.1017	.1637	1274	.1100	. 1933	.0776	6998					auc.	.1189	.1362	1111	.1336	1222	7160.	9146	16.0.	.0 150				***			2445	.0569	.0667	.0566	.0473	*****
5.5	SIGNA	· · · · · · · · · · · · · · · · · · ·	5332	. 6039	1182	\$118	. 9116	2666.	1.1600	1.247			•	SIGNAT			.1567									•	STGMAT		8940.	1250.			9560.	2260.	
SIGHA .	96	.0539		.0429	1910		. 9279	****	. 0177					20	.0326	9416.	*650	. 9378	. 9 37 3	1620 .	. 9258	2610.	. 8154			SIGHA	36			. 914.5	. 9174	. 019		.916.	
1.100	KQ/33	.2876	1893	.1296	9846	1950		9150	.0178	.0129				10.43	.1739	5651.		1686.	.0555	.0432	. 02557	1610.				1.100	K0/33	-	9996	. 0410	.0359	.0327	.0241		2000.
- 0118 HOL	27.75	.6996	2855.	***	2771	.2168	.1678	1276	.0662			-	110	\$1.72	1500	2644	3956	.3231	21.38	.1664	1597		24.0			PITCH PATIO	K1/32		.122.	.1288	11717	1611.	. 141	.0770	6010.
114 189	21449	.4839	. 5577	585	6.76	1911		2	. 5929	.916.				21.444				. 5732		.6584	. 6177	. 5761	. 5385				FFFIC		.4182		1055	5285	. 6175	. 6973	. 1686
PROPELLE NUMBER -4687	1.000×01	.5 839	.5.64		. 4327	.3942	. 1863	2746	.2366	.1965				1000301	.3787	.4381		593		.3704	. 2976	25575	.2134				106001		.1544	.1731	.2197	.2384	.7486	.2184	1743
-	*1001	.3146	.2735	.2261	. 2002	.1756	*161.						134046	10011	.1747	52822	2512.	.2964	2421	1920	1621.		. 1624			140044	*1001		. 0404			. 9464		.0794	
	,		.7000		2050	.9000				1.15				•		****		****		.4588			1.1500				,					. 40 10		1.0500	1.13

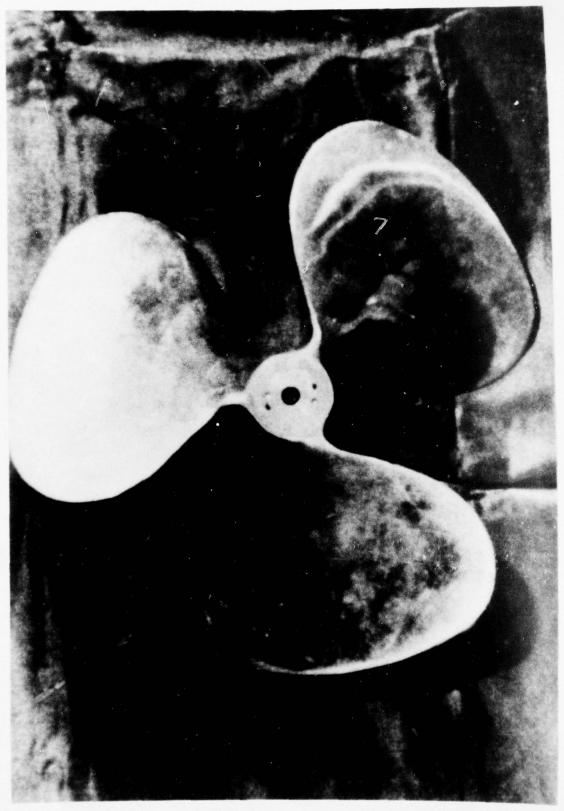
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Table 8

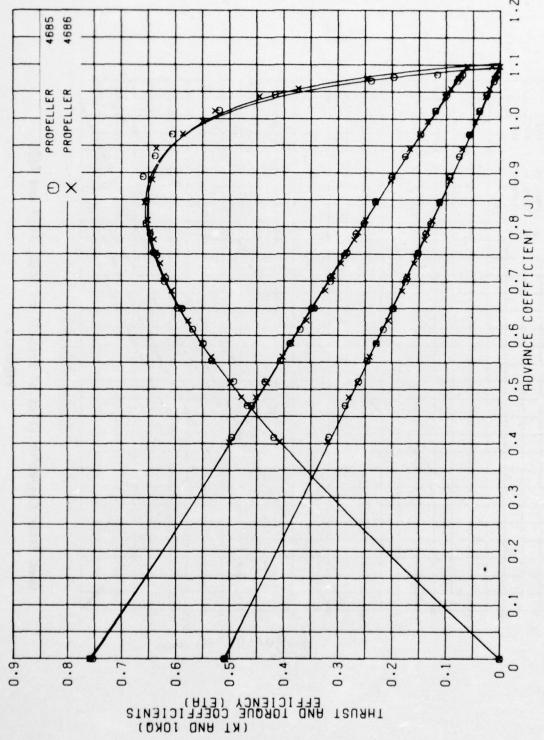
Cavitation Performance Characteristics of Propeller 4688

	MOM			PITCH 84110	. 1.30	SIGNA	• • • •			-			- 11CM 88710 -	1.30	SIGNA	• 3.0	
•	RTOUT	100031	EFFIC	\$1/18	KQ/J3	96	SIGHAP	TAUC	,	KTOUT	1.00001	21443	27/12	10.33	8	STENAT	7.800
.650	.3760	.6159	.4767	****	.797.			2540	****	1944						1	
.7000	.3500	.7653	. 5895	.7143	.2231	. 9658	. 5332	.2361	.758	. 3154	.7126	5263	1				
	. 3240	.7167	. 53%	.575	.1699	.0688	.6839	.2156		1062.	.6784	***		.1325		.38.	
					.1307	. 056	.6774	.1955		.2758	.6355	. 5070	.3017	.1835	.0524	. 3867	.1703
						.0513	. 7531	.1756		. 244	.5074	***		***		. 4381	1881.
	2200				26.00		. 6315	1961.		2222	. 5367		.2467	***		\$14.	.1391
						62.0.	. 9115	.1303		.1938	.4897	. 6350	.1930	***	301	. \$137	.1193
	****					2019.	. 9932	9021.	1.050.1	.1056	.4358	.63.	.1502	.1374	. 8336	. 5546	.1002
					2050.		1.0761	.1036	1.100	1384	.3861	.6200	.11.	2620.	. 629.		
				.1616		2068.	1.160	.0874	1.1500	.1126	.345		2500.	\$228.	. 1255		
					. 0234	.0249	1.2447	.0720	1.2800	:	.2 990	. 56.31		.8174	. 1219		
					2010	. 1230	1.329		1.2580		.2583	. 5059		132		. 7321	
		• • • • • • • • • • • • • • • • • • • •	. 3612		2410.	. 9196	1.4155	.0 +30	1.3980		.2167	. 4288	1920.	****	. 8192	. 7764	. 8263
		11.2	-				1.5011	. 8389	1.3580	. 8230	.1731	9682.	1210.				
						***	1.5866	.010.	1.40		.1200	1926.			•	1 30 .	:
				****		.211.	1.671	. 387.									
	1340M			PITCH RATED	1.300	STGHA	1.5			134004	*****					,	
										1	-				Siens	2.	
,	*1001	1940001	EFFIC	KT/ 32	K0/13	90	STGMAT	7.000	•	*****		*****			;		
												1	3571	****	3	SICHEL	-
	.2381	.5774	. 9250	.3720	.1128		.172	.1562	. 74 88	2668.	.2 786	. 4781	.176.		39		***
	200	. 5 963	. 5669	.3424	. 8961			.1594		1001.	.3041		.1789	****	.0294	. 8676	
		2,60					.2150	.1527		.1220	.3390	42	.1689	. 8549		. 8974	
	1067					224	1552.	*****		. 1 342	. 3624		.1687			.1075	***
1.0500				****							2105-	. 3000	.190		***	.1174	***
1.10	-1389	.3838	.6334		. 8788										***		
1.1500	.1121	.3362	.6183		. 0221	. 8258	1210										:
1.2000	6400.	2862.	. \$722	. 0611	. 0178	. 8214	. 3639	1050	1.1500								
1.2598	****	9462.	.918.	. 8429	. 6130		. 3661		1.7686	***	2775						
1.30		.2170	6244.	. 0275	6680.	.0192	. 3882	.8256	1.2488		.7 261	. 9787	1240	****			
1.34		.1757	. 3225	.0165	. 0071	.0121	. 4103	. 0142	1.1000	2000	.179.	70.					
	***	.1242	2090 .	. 0017	. 0045	***	.424.	. 101.	1.1929		.1 336	.2993	.010.	. 1154	2600		
									1.4444		****	. 11.52		. 1839		. 2162	
	PROPELLE	PROPELLEP NUMBER -4688	688 PITCH	CH RATTO .	1, 300	SIGHT .	••										
,	KTOUT	100001	EFFIC	K1/32	KQ/33	36	STGHAT	TAUC									
	. 9689	.1975	3926	. 8981	. 0386	. 916.	***										
		6102.					1	. 0431									
	. 0862	2678						****									
	2060.	.2718	5414	2460	8277												
1.0580	. 8983	.2878	.5786	1600	. 8249	. 9222		****									
1.1980	. 8973	.2928	.5834	.880	. 8219	22.6.		.8474									
1.1580	. 8986	.2826	.5000	. 96.46	.0106	. 0210	. 1073	. 0530									
1.2000	. 9789	.2688	. 5793		.0150	.4199	.1100	25.0.									
1.2400	. 9624	.2268	. 5473	.0344	.0116	. 0162	.1220	1660.									
	. 06.30		.4736	.828.		.0132	ź:	.0237									
	2200		9926	6216.		. 0103	1 366	2210.									
		0021.	. 0036	. 546.3		1826.		. 0074									

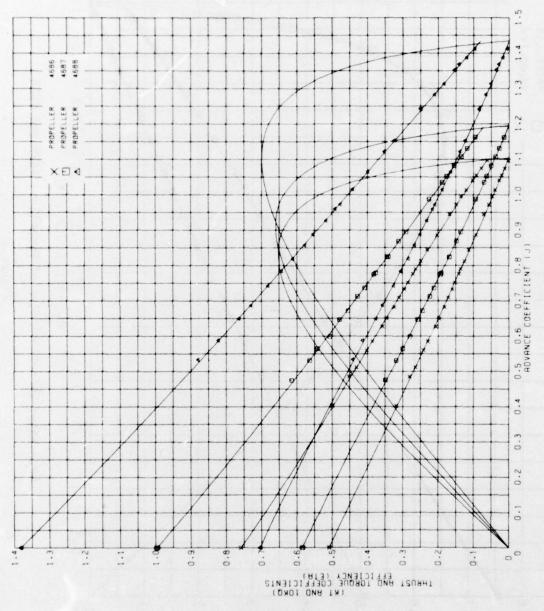
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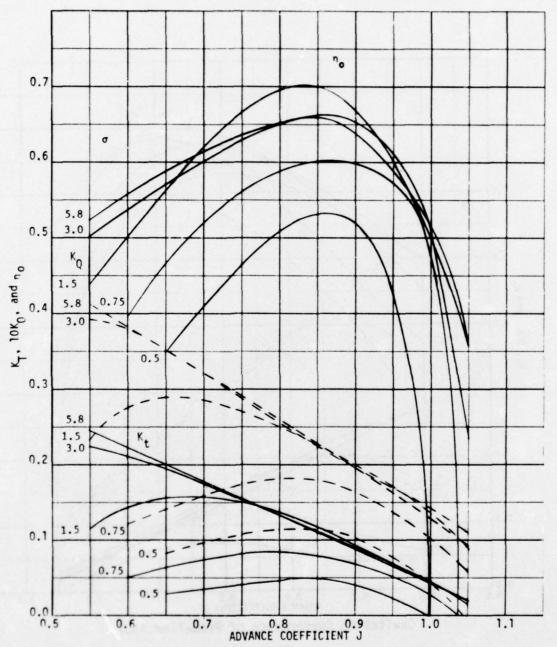
Photograph of a Commercial Propeller
Figure 1



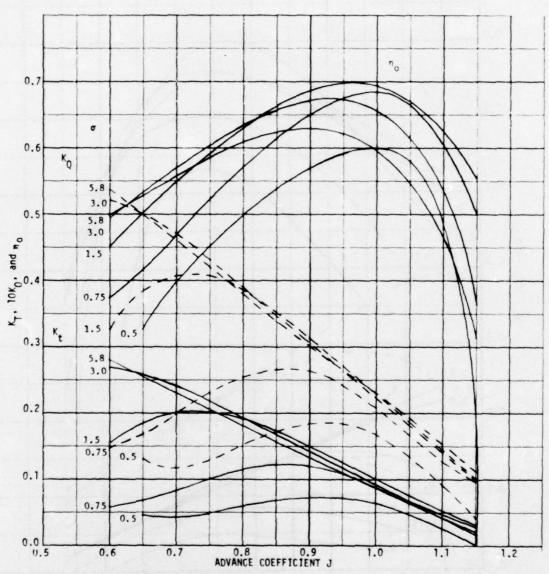
Open Water Characteristics of Propellers 4685 and 4686



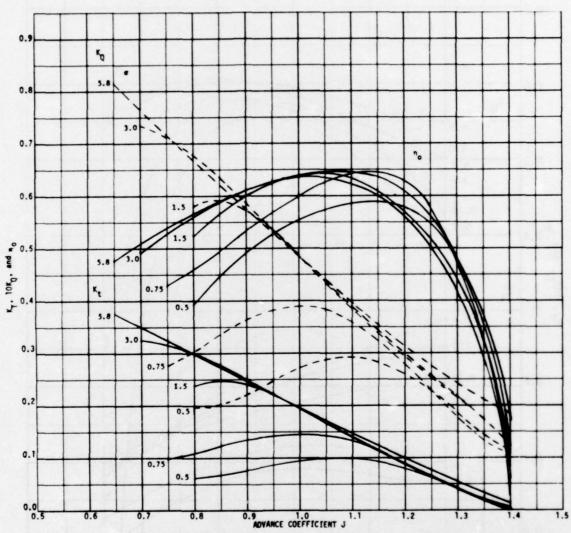
Open Water Characteristics of Propellers 4686, 4687 and 4688



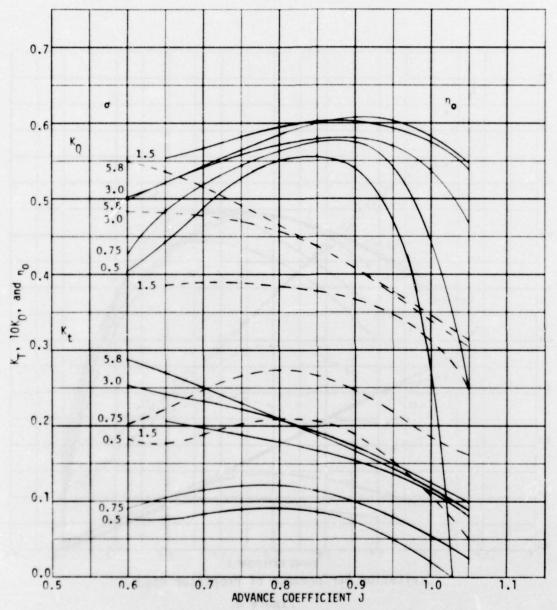
Cavitation Performance of Propeller 4686 Figure 4



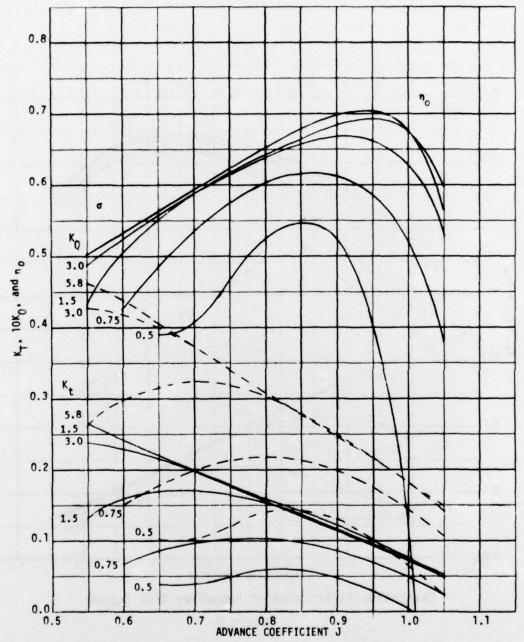
Cavitation Performance of Propeller 4687 Figure 5



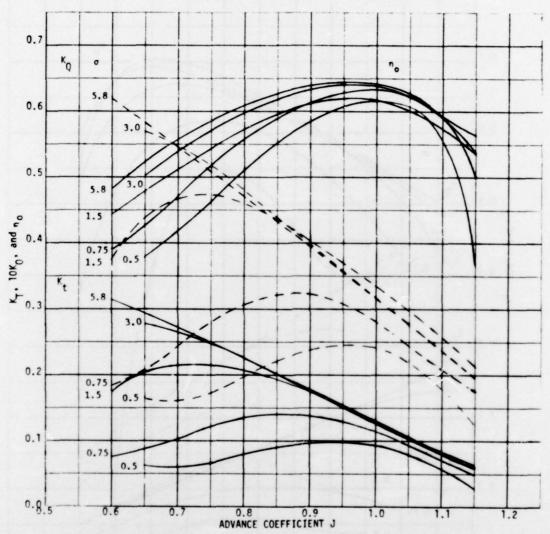
Cavitation Performance of Propeller 4688
Figure 6



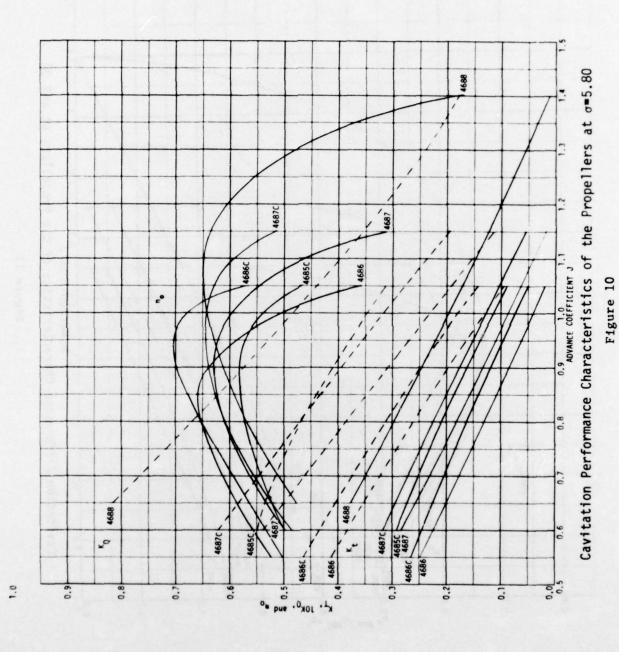
Cavitation Performance of Propeller 4685 Cupped Figure 7



Cavitation Performance of Propeller 4686 Cupped Figure 8



Cavitation Performance of Propeller 4687 Cupped Figure 9



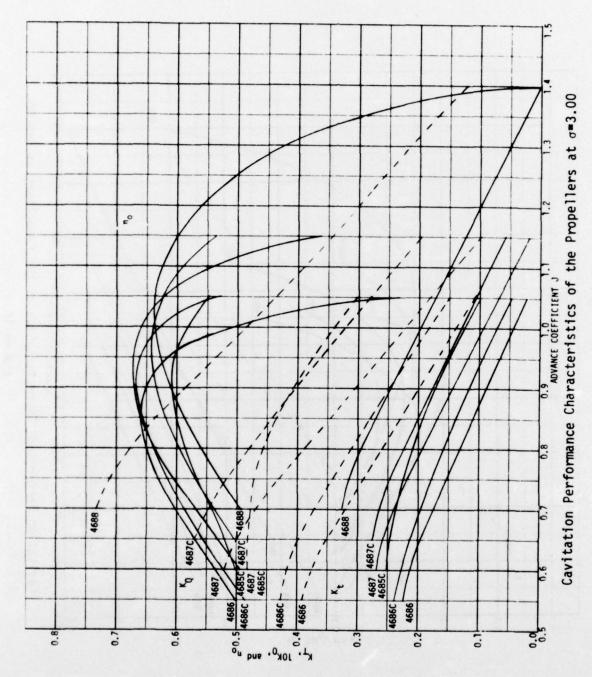
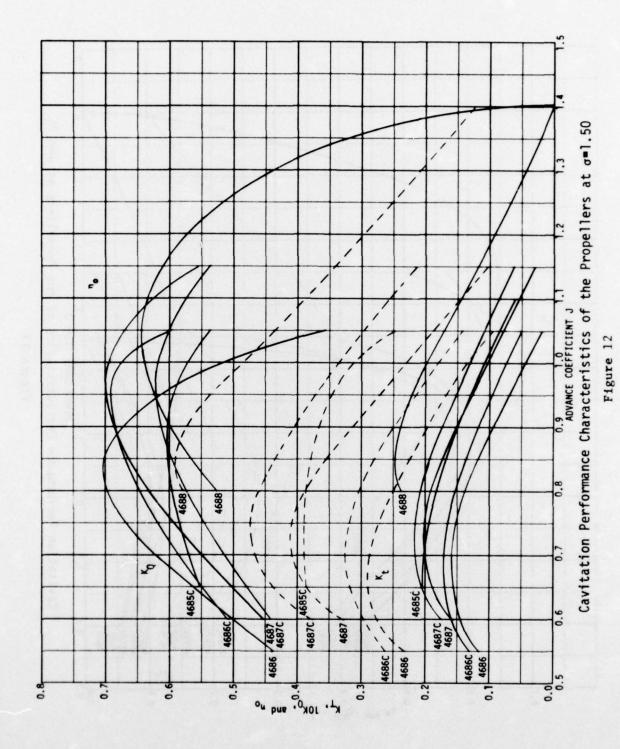
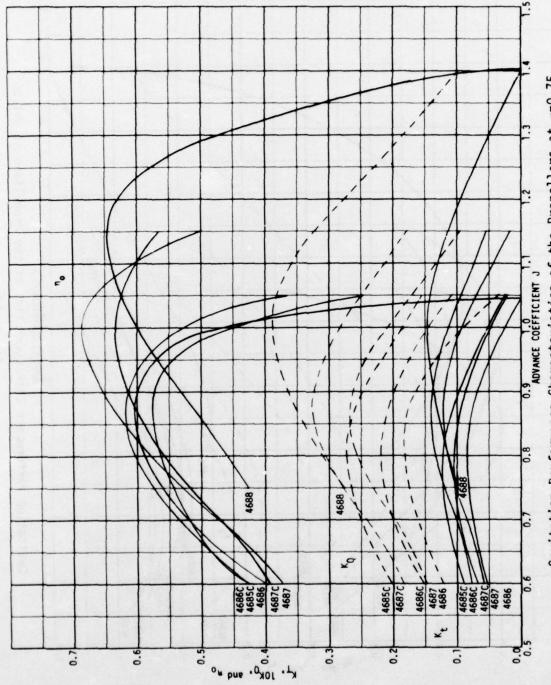
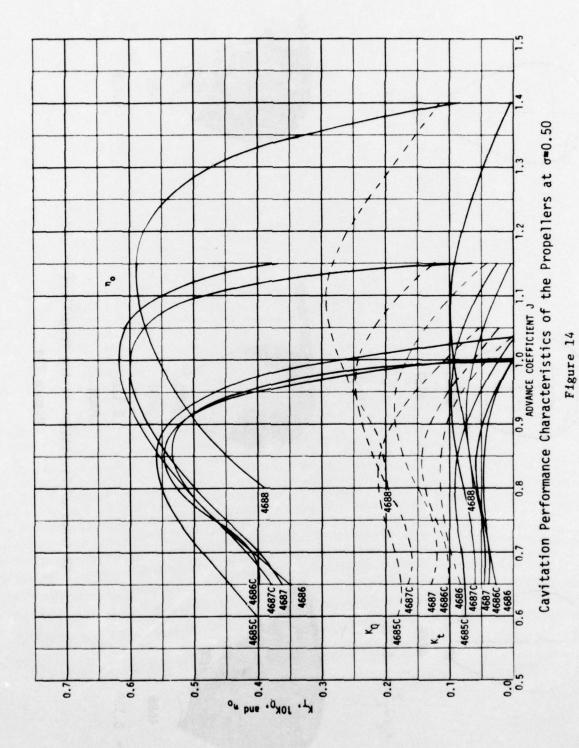


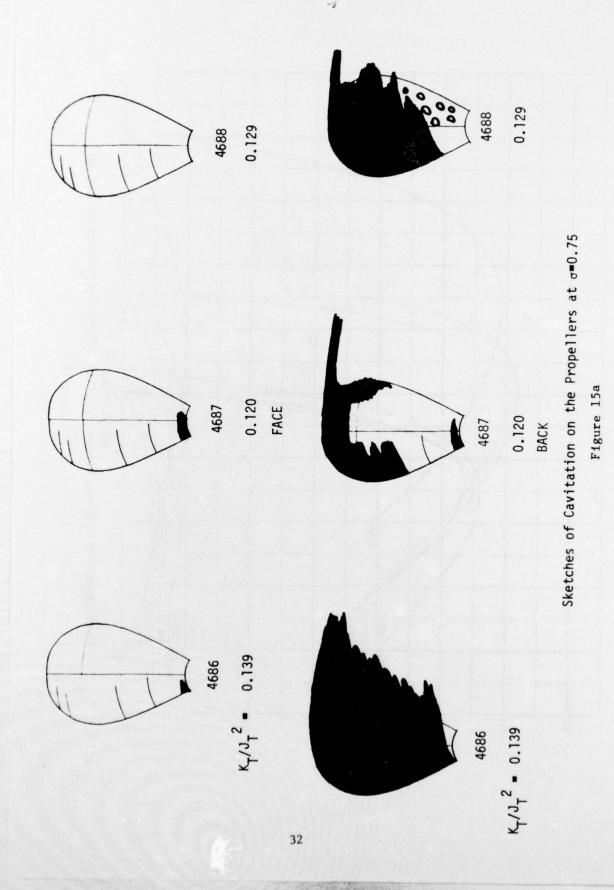
Figure 11

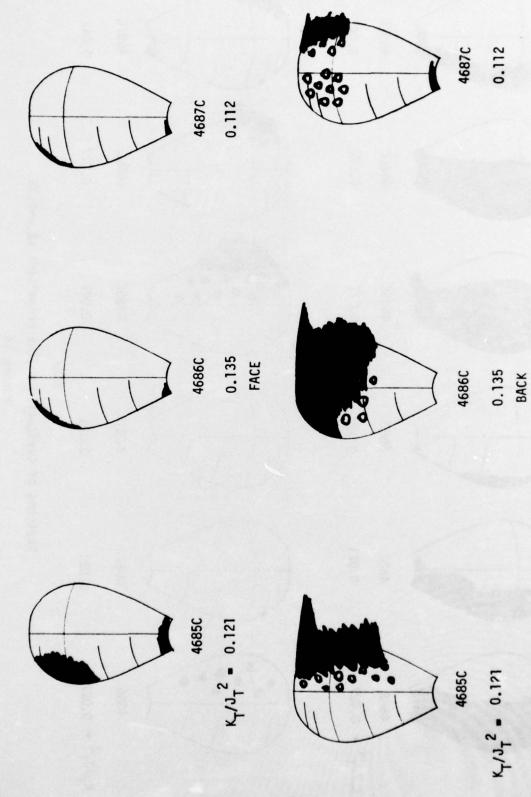




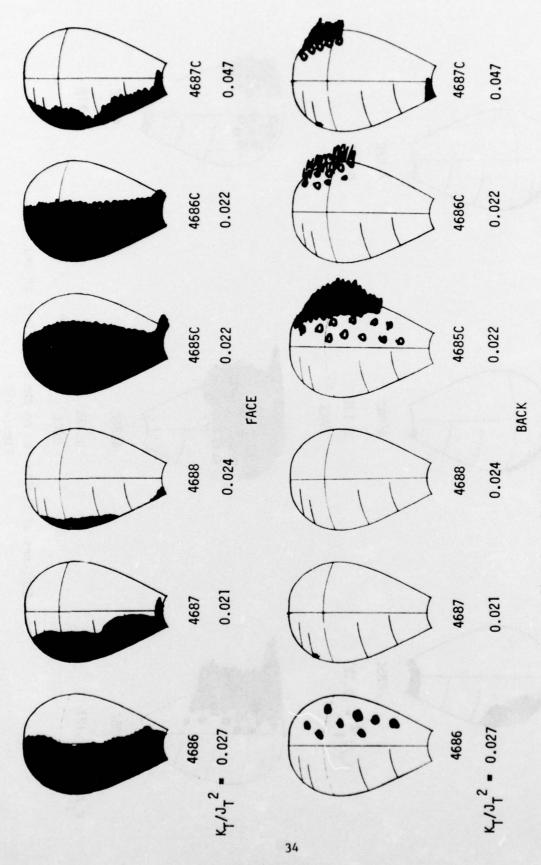
Cavitation Performance Characteristics of the Propellers at $\sigma = 0.75$



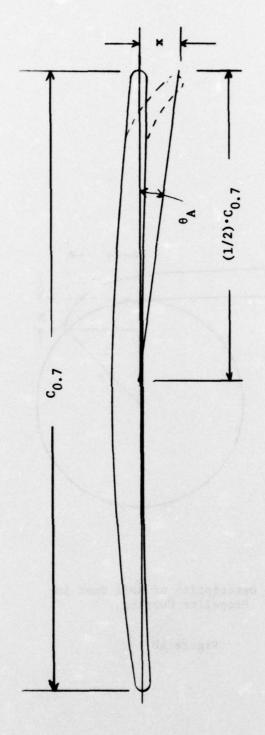




Sketches of Cavitation on the Propellers at σ=0.75 Figure 15b

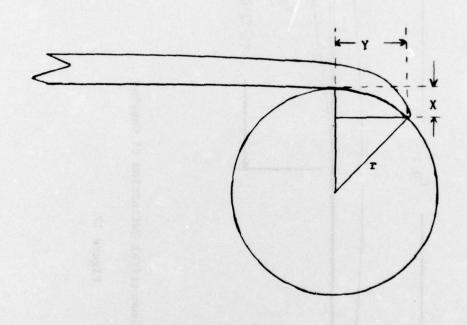


Sketches of Cavitation on the Propellers at σ=0.75



Geometrical Definition of Cupping

Figure 17



Geometrical Description of Ball Used in Propeller Cupping

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